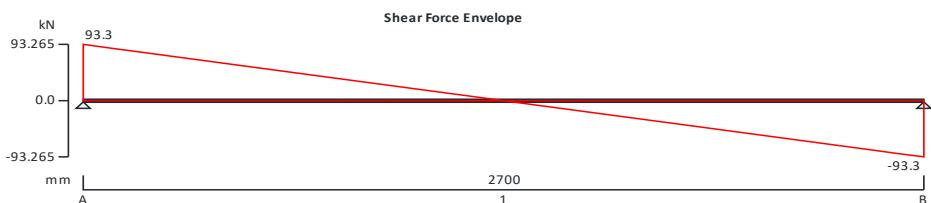
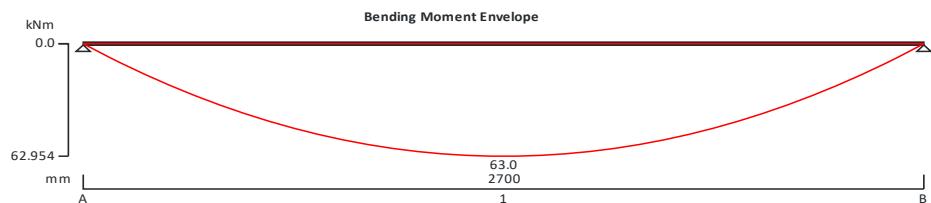
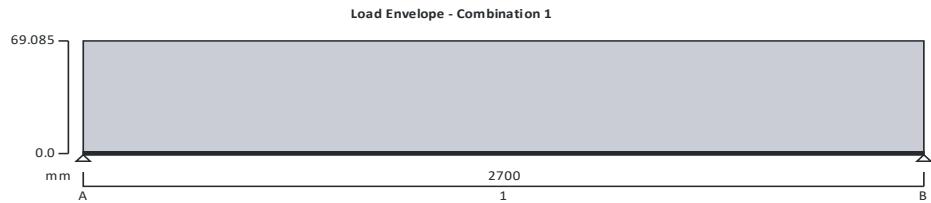


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	Calcs for	Mr Simon Heath			Start page no./Revision	1
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	SB	30/05/2023	DB	30/05/2023	SB	30/05/2023

## **STEEL BEAM ANALYSIS & DESIGN (EN1993-1-1:2005)**

**In accordance with EN1993-1-1:2005 incorporating Corrigenda February 2006 and April 2009 and the UK national annex**

TEDDS calculation version 3.0.14



## Support conditions

## Support A

Vertically restrained

## Support B

#### Vertically restrained

#### Rotationally free

## Applied loading

### Beam loads

Permanent self weight of beam  $\times 1$

Permanent full UDL 28.5 kN/m

Variable full UDL 20 kN/m

## Load combinations

### Load combination 1

## Support A

Permanent  $\times$  1.35

Variable  $\times 1.50$

Permanent × 1.35

Variable  $\times 1.50$

Support B

Permanent × 1.35

Variable  $\times 1.50$

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### Analysis results

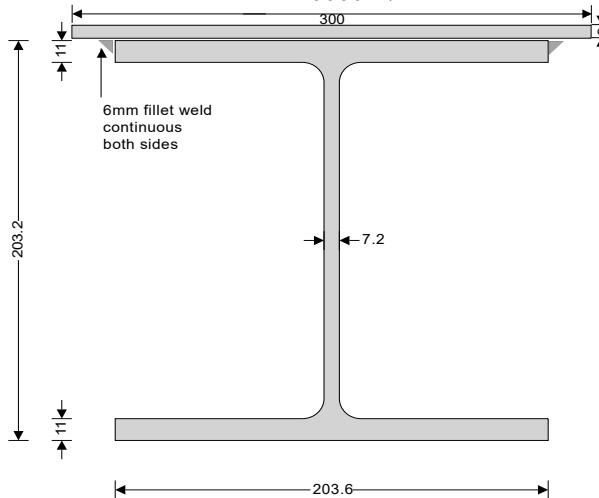
Maximum moment	$M_{\max} = 63 \text{ kNm}$	$M_{\min} = 0 \text{ kNm}$
Maximum shear	$V_{\max} = 93.3 \text{ kN}$	$V_{\min} = -93.3 \text{ kN}$
Deflection	$\delta_{\max} = 1.4 \text{ mm}$	$\delta_{\min} = 0 \text{ mm}$
Maximum reaction at support A	$R_{A\_max} = 93.3 \text{ kN}$	$R_{A\_min} = 93.3 \text{ kN}$
Unfactored permanent load reaction at support A	$R_{A\_Permanent} = 39.1 \text{ kN}$	
Unfactored variable load reaction at support A	$R_{A\_Variable} = 27 \text{ kN}$	
Maximum reaction at support B	$R_{B\_max} = 93.3 \text{ kN}$	$R_{B\_min} = 93.3 \text{ kN}$
Unfactored permanent load reaction at support B	$R_{B\_Permanent} = 39.1 \text{ kN}$	
Unfactored variable load reaction at support B	$R_{B\_Variable} = 27 \text{ kN}$	

### Section details

Section type	<b>UC 203x203x46 (BS4-1)</b>
Steel grade	<b>S275</b>

### EN 10025-2:2004 - Hot rolled products of structural steels

Nominal thickness of element	$t = \max(t_f, t_w) = 11.0 \text{ mm}$
Nominal yield strength	$f_y = 275 \text{ N/mm}^2$
Nominal ultimate tensile strength	$f_u = 410 \text{ N/mm}^2$
Modulus of elasticity	$E = 210000 \text{ N/mm}^2$



Flat 8x300mm plate, 2 No. 6mm fillet weld  
 Section depth, d, 300 mm  
 Section thickness, t, 8 mm  
 Mass of section, Mass, 18.8 kg/m  
 Area of section, A, 2400 mm<sup>2</sup>  
 Radius of gyration about y-axis,  $i_y$ , 86.603 mm  
 Radius of gyration about z-axis,  $i_z$ , 2.309 mm  
 Elastic section modulus about y-axis,  $W_{el,y}$ , 120000 mm<sup>3</sup>  
 Elastic section modulus about z-axis,  $W_{el,z}$ , 3200 mm<sup>3</sup>  
 Plastic section modulus about y-axis,  $W_{pl,y}$ , 180000 mm<sup>3</sup>  
 Plastic section modulus about z-axis,  $W_{pl,z}$ , 4800 mm<sup>3</sup>  
 Second moment of area about y-axis,  $I_y$ , 18000000 mm<sup>4</sup>  
 Second moment of area about z-axis,  $I_z$ , 12800 mm<sup>4</sup>

### Partial factors - Section 6.1

Resistance of cross-sections	$\gamma_{M0} = 1.00$
Resistance of members to instability	$\gamma_{M1} = 1.00$
Resistance of tensile members to fracture	$\gamma_{M2} = 1.10$

### Lateral restraint

Span 1 has lateral restraint at supports only

### Effective length factors

Effective length factor in major axis	$K_y = 1.000$
Effective length factor in minor axis	$K_z = 1.000$
Effective length factor for torsion	$K_{LT,A} = 1.000$
	$K_{LT,B} = 1.000$

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### Classification of cross sections - Section 5.5

$$\varepsilon = \sqrt{[235 \text{ N/mm}^2 / f_y]} = \mathbf{0.92}$$

### Internal compression parts subject to bending and compression - Table 5.2 (sheet 1 of 3)

Width of section

$$c = d = \mathbf{160.8 \text{ mm}}$$

$$\alpha = \min([h / 2 + N_{Ed} / (2 \times t_w \times f_y) - (t_f + r)] / c, 1) = \mathbf{0.971}$$

$$c / t_w = 24.2 \times \varepsilon \leq 396 \times \varepsilon / (13 \times \alpha - 1) \quad \text{Class 1}$$

### Outstand flanges - Table 5.2 (sheet 2 of 3)

Width of section

$$c = (b - t_w - 2 \times r) / 2 = \mathbf{88 \text{ mm}}$$

$$c / t_f = 8.7 \times \varepsilon \leq 9 \times \varepsilon \quad \text{Class 1}$$

**Section is class 1**

### Check shear - Section 6.2.6

Height of web

$$h_w = h - 2 \times t_f = \mathbf{181.2 \text{ mm}}$$

Shear area factor

$$\eta = \mathbf{1.000}$$

$$h_w / t_w < 72 \times \varepsilon / \eta$$

**Shear buckling resistance can be ignored**

Design shear force

$$V_{Ed} = \max(\text{abs}(V_{\max}), \text{abs}(V_{\min})) = \mathbf{93.3 \text{ kN}}$$

Shear area - cl 6.2.6(3)

$$A_v = \max(A - 2 \times b \times t_f + (t_w + 2 \times r) \times t_f, \eta \times h_w \times t_w) = \mathbf{1698 \text{ mm}^2}$$

Design shear resistance - cl 6.2.6(2)

$$V_{c,Rd} = V_{pl,Rd} = A_v \times (f_y / \sqrt{3}) / \gamma_{M0} = \mathbf{269.5 \text{ kN}}$$

**PASS - Design shear resistance exceeds design shear force**

### Check bending moment major (y-y) axis - Section 6.2.5

Design bending moment

$$M_{Ed} = \max(\text{abs}(M_{s1,\max}), \text{abs}(M_{s1,\min})) = \mathbf{63 \text{ kNm}}$$

Design bending resistance moment - eq 6.13

$$M_{c,Rd} = M_{pl,Rd} = W_{pl,y} \times f_y / \gamma_{M0} = \mathbf{136.8 \text{ kNm}}$$

### Slenderness ratio for lateral torsional buckling

Correction factor - Table 6.6

$$k_c = \mathbf{0.94}$$

$$C_1 = 1 / k_c^2 = \mathbf{1.132}$$

$$g = \sqrt{[1 - (l_z / l_y)]} = \mathbf{0.813}$$

$$v = \mathbf{0.3}$$

$$G = E / [2 \times (1 + v)] = \mathbf{80769 \text{ N/mm}^2}$$

$$L = 1.0 \times L_{s1} = \mathbf{2700 \text{ mm}}$$

$$M_{cr} = C_1 \times \pi^2 \times E \times I_z / (L^2 \times g) \times \sqrt{[I_w / I_z + L^2 \times G \times I_t / (\pi^2 \times E \times I_z)]} = \mathbf{706.6 \text{ kNm}}$$

Slenderness ratio for lateral torsional buckling

$$\bar{\lambda}_{LT} = \sqrt{(W_{pl,y} \times f_y / M_{cr})} = \mathbf{0.44}$$

Limiting slenderness ratio

$$\bar{\lambda}_{LT,0} = \mathbf{0.4}$$

**$\bar{\lambda}_{LT} > \bar{\lambda}_{LT,0}$  - Lateral torsional buckling cannot be ignored**

### Design resistance for buckling - Section 6.3.2.1

Buckling curve - Table 6.5

$$b$$

Imperfection factor - Table 6.3

$$\alpha_{LT} = \mathbf{0.34}$$

Correction factor for rolled sections

$$\beta = \mathbf{0.75}$$

LTB reduction determination factor

$$\phi_{LT} = 0.5 \times [1 + \alpha_{LT} \times (\bar{\lambda}_{LT} - \bar{\lambda}_{LT,0}) + \beta \times \bar{\lambda}_{LT}^2] = \mathbf{0.579}$$

LTB reduction factor - eq 6.57

$$\chi_{LT} = \min(1 / [\phi_{LT} + \sqrt{(\phi_{LT}^2 - \beta \times \bar{\lambda}_{LT}^2)}], 1, 1 / \bar{\lambda}_{LT}^2) = \mathbf{0.984}$$

Modification factor

$$f = \min(1 - 0.5 \times (1 - k_c) \times [1 - 2 \times (\bar{\lambda}_{LT} - 0.8)^2], 1) = \mathbf{0.978}$$

Modified LTB reduction factor - eq 6.58

$$\chi_{LT,mod} = \min(\chi_{LT} / f, 1) = \mathbf{1.000}$$

Design buckling resistance moment - eq 6.55

$$M_{b,Rd} = \chi_{LT,mod} \times W_{pl,y} \times f_y / \gamma_{M1} = \mathbf{136.8 \text{ kNm}}$$

**PASS - Design buckling resistance moment exceeds design bending moment**

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#### Check compression - Section 6.2.4

Design compression force  $N_{Ed} = \mathbf{300}$  kN

Design resistance of section - eq 6.10  $N_{c,Rd} = N_{pl,Rd} = A \times f_y / \gamma_{M0} = \mathbf{1615.1}$  kN

#### Slenderness ratio for major (y-y) axis buckling

Critical buckling length  $L_{cr,y} = L_{s1} \times K_y = \mathbf{2700}$  mm

Critical buckling force  $N_{cr,y} = \pi^2 \times E_{SEC3} \times I_y / L_{cr,y}^2 = \mathbf{12986.7}$  kN

Slenderness ratio for buckling - eq 6.50  $\bar{\lambda}_y = \sqrt{[A \times f_y / N_{cr,y}]} = \mathbf{0.353}$

#### Design resistance for buckling - Section 6.3.1.1

Buckling curve - Table 6.2 b

Imperfection factor - Table 6.1  $\alpha_y = \mathbf{0.34}$

Buckling reduction determination factor  $\phi_y = 0.5 \times [1 + \alpha_y \times (\bar{\lambda}_y - 0.2) + \bar{\lambda}_y^2] = \mathbf{0.588}$

Buckling reduction factor - eq 6.49  $\chi_y = \min(1 / [\phi_y + \sqrt{(\phi_y^2 - \bar{\lambda}_y^2)}], 1) = \mathbf{0.944}$

Design buckling resistance - eq 6.47  $N_{b,y,Rd} = \chi_y \times A \times f_y / \gamma_{M1} = \mathbf{1525.4}$  kN

**PASS - Design buckling resistance exceeds design compression force**

#### Slenderness ratio for minor (z-z) axis buckling

Critical buckling length  $L_{cr,z} = L_{s1\_seg1} \times K_z = \mathbf{2700}$  mm

Critical buckling force  $N_{cr,z} = \pi^2 \times E_{SEC3} \times I_z / L_{cr,z}^2 = \mathbf{4401.7}$  kN

Slenderness ratio for buckling - eq 6.50  $\bar{\lambda}_z = \sqrt{[A \times f_y / N_{cr,z}]} = \mathbf{0.606}$

#### Design resistance for buckling - Section 6.3.1.1

Buckling curve - Table 6.2 c

Imperfection factor - Table 6.1  $\alpha_z = \mathbf{0.49}$

Buckling reduction determination factor  $\phi_z = 0.5 \times [1 + \alpha_z \times (\bar{\lambda}_z - 0.2) + \bar{\lambda}_z^2] = \mathbf{0.783}$

Buckling reduction factor - eq 6.49  $\chi_z = \min(1 / [\phi_z + \sqrt{(\phi_z^2 - \bar{\lambda}_z^2)}], 1) = \mathbf{0.782}$

Design buckling resistance - eq 6.47  $N_{b,z,Rd} = \chi_z \times A \times f_y / \gamma_{M1} = \mathbf{1263}$  kN

**PASS - Design buckling resistance exceeds design compression force**

#### Check torsional and torsional-flexural buckling - Section 6.3.1.4

Torsional buckling length factor  $K_T = \mathbf{1.00}$

Torsional buckling length  $L_{cr,T} = \max(L_{s1}, L_{s1\_seg1}) \times K_T = \mathbf{2700}$  mm

Distance from shear centre to centroid in y axis  $y_0 = \mathbf{0.0}$  mm

Distance from shear centre to centroid in z axis  $z_0 = \mathbf{0.0}$  mm

Radius of gyration  $i_0 = \sqrt{i_y^2 + i_z^2} = \mathbf{102.0}$  mm

Elastic critical torsional buckling force  $N_{cr,T} = 1 / i_0^2 \times [G \times I_t + \pi^2 \times E_{SEC3} \times I_w / L_{cr,T}^2] = \mathbf{5621.9}$  kN

Torsion factor  $\beta_T = 1 - (y_0 / i_0)^2 = \mathbf{1.000}$

Elastic critical torsional-flexural buckling force

$$N_{cr,TF} = N_{cr,y} / (2 \times \beta_T) \times [1 + N_{cr,T} / N_{cr,y} - \sqrt{[(1 - N_{cr,T} / N_{cr,y})^2 + 4 \times (y_0 / i_0)^2 \times N_{cr,T} / N_{cr,y}]}] = \mathbf{5621.9}$$
 kN

Elastic critical buckling force  $N_{cr} = \min(N_{cr,T}, N_{cr,TF}) = \mathbf{5621.9}$  kN

Slenderness ratio for torsional buckling - eq 6.52  $\bar{\lambda}_T = \sqrt{[A \times f_y / N_{cr}]} = \mathbf{0.536}$

#### Design resistance for buckling - Section 6.3.1.1

Buckling curve - Table 6.2 c

Imperfection factor - Table 6.1  $\alpha_T = \mathbf{0.49}$

Buckling reduction determination factor  $\phi_T = 0.5 \times [1 + \alpha_T \times (\bar{\lambda}_T - 0.2) + \bar{\lambda}_T^2] = \mathbf{0.726}$

Buckling reduction factor - eq 6.49  $\chi_T = \min(1 / [\phi_T + \sqrt{(\phi_T^2 - \bar{\lambda}_T^2)}], 1) = \mathbf{0.823}$

Design buckling resistance - eq 6.47  $N_{b,T,Rd} = \chi_T \times A \times f_y / \gamma_{M1} = \mathbf{1328.7}$  kN

**PASS - Design buckling resistance exceeds design compression force**

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### Combined bending and axial force - Section 6.2.9

Normal force to plastic resistance force ratio  $n = N_{Ed} / N_{Pl,Rd} = \mathbf{0.19}$   
 Web area to gross area ratio  $a_w = \min((A - 2 \times b \times t_f) / A, 0.5) = \mathbf{0.24}$   
 Design plastic moment resistance - eq 6.13  $M_{Pl,Rd} = W_{pl,y} \times f_y / \gamma_{M0} = \mathbf{136.8} \text{ kNm}$   
 Reduced plastic moment resistance - eq 6.36  $M_{N,Rd} = M_{Pl,Rd} \times \min((1 - n) / (1 - 0.5 \times a_w), 1) = \mathbf{126.4} \text{ kNm}$

**PASS - Reduced bending resistance moment exceeds design bending moment**

### Check combined bending and compression - Section 6.3.3

Equivalent uniform moment factors - Table B.3  $M_{hy} = \mathbf{0} \text{ kNm}$   
 $M_{sy} = \mathbf{63} \text{ kNm}$   
 $\psi_y = \mathbf{1.000}$   
 $\alpha_{hy} = M_{hy} / M_{sy} = \mathbf{0.000}$   
 $C_{my} = 0.95 + 0.05 \times \alpha_{hy} = \mathbf{0.950}$   
 $M_{hz} = \mathbf{0} \text{ kNm}$   
 $M_{sz} = \mathbf{0} \text{ kNm}$   
 $\psi_z = \mathbf{1.000}$   
 $C_{mz} = 0.6 + 0.4 \times \psi_z = \mathbf{1.000}$   
 $M_{hLT} = \mathbf{0} \text{ kNm}$   
 $M_{sLT} = \mathbf{63} \text{ kNm}$   
 $\psi_{LT} = \mathbf{1.000}$   
 $\alpha_{hLT} = M_{hLT} / M_{sLT} = \mathbf{0.000}$   
 $C_{mLT} = 0.95 + 0.05 \times \alpha_{hLT} = \mathbf{0.950}$

### Interaction factors $k_{ij}$ for members susceptible to torsional deformations - Table B.2

Characteristic moment resistance  $M_{Rk} = W_{pl,y} \times f_y = \mathbf{136.8} \text{ kNm}$   
 Characteristic resistance to normal force  $N_{Rk} = A \times f_y = \mathbf{1615.1} \text{ kN}$   
 Interaction factors  $k_{yy} = C_{my} \times [1 + \min(\bar{\lambda}_y - 0.2, 0.8) \times N_{Ed} / (\chi_y \times N_{Rk} / \gamma_{M1})] = \mathbf{0.979}$   
 $k_{zy} = 1 - 0.1 \times \min(1, \bar{\lambda}_z) \times N_{Ed} / ((C_{mLT} - 0.25) \times \chi_z \times N_{Rk} / \gamma_{M1}) = \mathbf{0.979}$   
 $N_{Ed} / (\chi_y \times N_{Rk} / \gamma_{M1}) + k_{yy} \times M_{Ed} / (\chi_{LT} \times M_{Rk} / \gamma_{M1}) = \mathbf{0.654}$   
 $N_{Ed} / (\chi_z \times N_{Rk} / \gamma_{M1}) + k_{zy} \times M_{Ed} / (\chi_{LT} \times M_{Rk} / \gamma_{M1}) = \mathbf{0.695}$

**PASS - Combined bending and compression checks are satisfied**

### Check vertical deflection - Section 7.2.1

Consider deflection due to variable loads  
 Limiting deflection  $\delta_{lim} = L_{s1} / 360 = \mathbf{7.5} \text{ mm}$   
 Maximum deflection span 1  $\delta = \max(\text{abs}(\delta_{max}), \text{abs}(\delta_{min})) = \mathbf{1.443} \text{ mm}$

**PASS - Maximum deflection does not exceed deflection limit**